## NATURALLY-TRANSITIONING RATE-TO-FORCE CONTROL IN FREE AND CONSTRAINED MOTION

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## **ABSTRACT**

The Naturally-Transitioning Rate-to-Force Controller (*NTRFC*) is presented for teleoperation of manipulators. Our goal is to provide a single controller which handles free motion, constrained motion, and the transition in-between without any artificial changes. In free motion the displacement of the master device (via the human operator's hand) is proportional to the commanded Cartesian rate of the manipulator. In contact the displacement of the human operator's hand is proportional to the wrench (force/moment) exerted on the environment by the manipulator. The transition between free rate motion and applied-wrench contact with the environment requires no changes in control mode or gains and hence is termed natural. Furthermore, in contact, if the master enables force reflection, the wrench of the human operator's hand exerted on the master is proportional to the wrench exerted on the environment by the manipulator. This article demonstrates the *NTRFC* concept via a simple 1-dof model and then discusses experimental implementation and results from a Merlin manipulator teleoperated via the force-reflecting PHANTOM interface.

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#### 1. INTRODUCTION

Two fundamental problems in manipulator teleoperation are free motions in Cartesian space and the contacting of the environment during task performance. Resolved-rate control (Whitney, 1969) is popular for free motion. However, rate control is not widely implemented in practice, perhaps due to the difficulty of rate control when the manipulator contacts the environment. If a constant rate is commanded while the manipulator contacts the environment, commanded joint angles integrate while actual joint angles are constrained until an unacceptably large wrench (force/moment vector) is exerted by the manipulator. The general problem of changing from free to constrained motion is termed the "phase transition problem" by Tarn et al. (1996) and is a current topic of research.

Raibert and Craig (1981) presented a hybrid control method wherein some Cartesian axes are controlled in position while the remaining axes are force controlled. While this method is effective in practical tasks, one must choose either position or force on each Cartesian axis. Whitney (1985) reviews various force control architectures. Hogan (1985) presented an impedance controller where the behavior of a manipulator is controlled to mimic a 6-degree-of-freedom (dof) Cartesian *m-c-k* system. The controller discussed in this article, when in contact with the environment, is analogous to an impedance controller with only a damping term. In free motion, the controller acts like a rate controller. Most importantly, during the phase transition (the impact), the controller does not have to switch modes of control.

Colbaugh et al. (1993) present an adaptive scheme for controlling the end-effector impedance of robot manipulators in contact; however, an explicit control mode change is required for free motion. Hyde and Cutkosky (1994) experimentally evaluate several methods for controlling the transition from free motion to constrained motion, using a one-axis impact testbed. Yao and Tomizuka (1995) present an adaptive motion and force controller for manipulators with uncertainties in both the robot and contact surfaces. Vukobratovic et al. (1996) consider the problem of simultaneous stabilization of both the

robot motion and interaction force in Cartesian space after contact in robotic tasks. Tarn et al. (1996) use an event-driven switching control strategy for robot impact control and force regulation where the instant of impact must be known. Tarn et al. state that control of manipulator impact and contact is an important current research area. They present an excellent literature review of the subject: all of the reviewed methods require an artificial control mode change in the transition from free motion to contact.

The objective of the current article is to present a promising controller for solving this transition problem in teleoperation. Human operator effectiveness studies are beyond the scope of this article. There is a rich literature in telemanipulation and human operator/task performance effectiveness under various control algorithms, including force-reflection (e.g. see Das et al., 1992 and Sheridan, 1992).

The current article presents a method for dealing with the contact problem in teleoperation of manipulators, the Naturally-Transitioning Rate-to-Force Controller (*NTRFC*). The goal of this work is to provide a single controller which requires no artificial control modes changes in the transition from free motion to contact. Under commands from the operator via a master device, in free motion the manipulator moves with rate control, while in contact with the environment the wrench exerted on the environment is controlled. No artificial control mode or gain parameter changes are required so the transition is termed natural. A wrist-mounted force/torque (F/T) sensor and Force/Moment Accommodation (*FMA*) algorithm are required. Rate and *FMA* are active on all Cartesian axes simultaneously so no hybrid scheme is necessary. Since there are no artificial mode changes required, the threshold of contact is unimportant. A force-reflecting master may be used in conjunction with the *NTRFC* to increase the operator's sense of *telepresence* (the force-reflecting master is optional).

The *NTRFC* was originally developed experimentally at NASA Langley Research Center (Williams et al., 1996). It was implemented experimentally and proven very effective in completion of representative space telerobotics tasks. We have expanded and implemented the *NTRFC* during the summers of 1997 and 1998 in an experimental system at Wright-Patterson AFB. The goal of this article

is to demonstrate the *NTRFC* concept via a modeling example and to present experimental results. Section 2 presents a description of the *NTRFC* algorithm, Section 3 presents a simple modeling example to demonstrate the concept, and Section 4 covers experimental implementation and results.

## 2. NTRFC DESCRIPTION

This section presents the Naturally-Transitioning Rate-to-Force Controller (*NTRFC*) concept. It is applicable to teleoperation of a manipulator with wrist-mounted force/torque (F/T) sensor, Cartesian rate inputs, and tasks requiring contact with the environment.

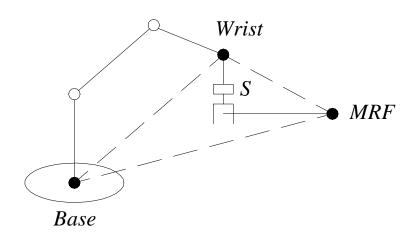


Figure 1. Manipulator Coordinate Frames

Figure 1 shows the manipulator coordinate frames used. The *Base* and *Wrist* frames are familiar. The Moving Reference Frame (MRF, denoted  $\{M\}$ ) is the user-defined control frame.  $\{S\}$  is the F/T sensor frame. Figure 2 shows the NTRFC high-level control diagram. The two basic active ingredients are the resolved rate and force/moment accommodation (FMA) algorithms, described below. In addition, master devices enabling force-reflection can be used effectively, though this is not required in the NTRFC.

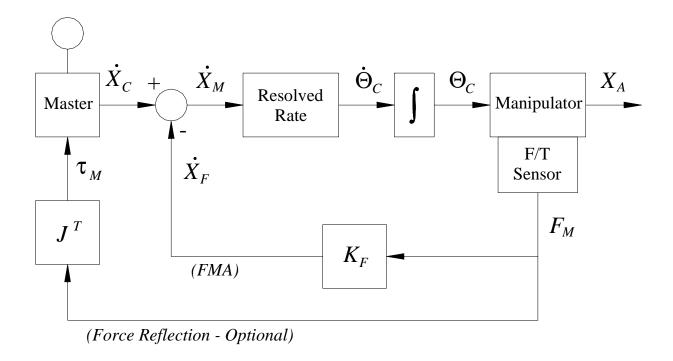


Figure 2. NTRFC Control Diagram

The resolved-rate algorithm is based on Whitney (1969). The manipulator Jacobian matrix  $J_M$  maps the joint rates to Cartesian rates  $\dot{X}_M = J_M \dot{\Theta}_C$ . In teleoperation, Cartesian rate inputs in  $\{M\}$ ,  $\dot{X}_M = \{v_M \ \omega_M\}^T$  are commanded by the human using the master device. The rate equation is solved at each control step to calculate the instantaneous joint rates necessary to achieve the commanded Cartesian rate:  $\dot{\Theta}_C = J_M^{-1} \dot{X}_M$ . (Gaussian elimination is a more efficient and robust solution procedure than the matrix inverse). The NTRFC is also applicable to teleoperation of kinematically-redundant manipulators by replacing the matrix inverse with the matrix pseudoinverse. However, this case is not presented in the current article. The joint rates are integrated to commanded joint angles  $\Theta_C$ . The manipulator controller continuously servos to  $\Theta_C$ ; the actual Cartesian pose  $X_A$  results.

The resolved-rate algorithm is sensitive to singularities. When the determinant of  $J_M$  approaches zero, the matrix inverse is replaced by a matrix pseudoinverse based on Singular Value Decomposition (SVD) until the manipulator is through the singular neighborhood.

If the manipulator is in contact with its environment, there are constraints on  $X_A$  (the actual Cartesian pose in Fig. 2) and a Cartesian contact wrench exists. In this article, wrench indicates a six-dof force/moment vector. A force/moment accommodation (FMA) algorithm has been implemented (simultaneously with the resolved-rate control) to automatically achieve zero wrench in order to minimize binding wrenches of contact during teleoperation. As discussed in the Introduction, this FMA algorithm is similar to an impedance controller (Hogan, 1985) with only the damping term. However, unlike the general impedance controller, the FMA algorithm is enabled during free and constrained motion.

A six-dof wrist-mounted F/T sensor reads the contact wrench  $F_S = \{f_S \mid m_S\}^T$  at each control step. The weight and moment of the end-effector and payload must be subtracted from  $F_S$ , accounting for the configuration. In addition, the inertial wrench due to end-effector accelerations should also be subtracted to avoid spurious FMA in free motion. This inertial component is difficult to determine online due to the lack of acceleration feedback. However, in experimental implementation, we used a wrench deadband to avoid noise problems; this also masked the inertial loading at the low rates of acceleration performed experimentally. The feedback wrench must be in  $\{M\}$  so the modified sensor reading is transformed to the MRF wrench  $F_M$  via rigid body wrench transformations (Craig, 1989). This wrench is converted to a Cartesian rate  $\dot{X}_F = K_F F_M$  and sent to the summing junction in Fig. 2. The diagonal gain matrix  $K_F$  has units m/Ns and rad/Nms for translational and rotational terms, respectively. The subscript F denotes FMA.

In the Naturally-Transitioning Rate-to-Force Controller (*NTRFC*), the resolved-rate algorithm is concurrently in operation with the *FMA* algorithm, for all Cartesian axes (no hybrid scheme is necessary). The overall resolved-rate input is the difference of the human-commanded rate and the *FMA* rate,  $\dot{X}_M = \dot{X}_C - \dot{X}_F$ . As the manipulator end-effector approaches a wall in the environment, the

Cartesian rates commanded by the operator attempt to command motion through the wall, but the *FMA* controller commands a reverse motion to exert zero wrench. Therefore, an equilibrium condition is entered, where the displacement of the master device is proportional to the exerted Cartesian contact wrench. When the optional force-reflection is enabled by the master device, the wrench of the human hand on the master is proportional to the wrench exerted by the manipulator on the environment. The *NTRFC* automatically corrects misalignments so insertion tasks can be completed with minimum contact wrenches. If no *FMA* is used, it is difficult to complete tasks since the manipulator is "blind" in a wrench sense.

The teleoperated system behaves as a rate controller in free motion and as a simplified impedance controller in contact. The transition requires no mode changes, logical switches, or gain changes in the controller software or hardware and thus is termed a natural transition. The transition is a consequence of the physics of manipulator contact with the environment when using the control architecture of Fig. 2. Assuming a well-calibrated F/T sensor with sufficient wrench deadband, the *NTRFC* does not care when the moment of contact occurs. The *FMA* algorithm is enabled continuously (on all Cartesian axes, simultaneously with rate control on all Cartesian axes), but only generates non-zero  $\dot{X}_F$  in contact. The next section presents a simple model to demonstrate the *NTRFC* concept.

## 3. NTRFC MODELING

This section presents dynamics and control modeling for a simple 1-dof 1*P* device in motion under the *NTRFC*. A basic *NTRFC* design procedure is outlined; this same procedure is expanded for real-world hardware in the next section. Since the free-motion to contact transition is a natural one, we must obtain desirable performance with only one set of gains and software control mode.

In hardware implementation the manipulator dynamics and environment characteristics are provided by the real world. For modeling, the *NTRFC* diagram in Fig. 2 must be modified to model these real world effects: 1) manipulator forward dynamics and forward kinematics must be calculated; 2) an environment model predicts the contact wrench  $F_M$  assuming a perfect F/T sensor and lumped environment characteristics; and 3) the operator-commanded manipulator Cartesian rate  $\dot{x}_c$  is assumed to come from a fictitious teleoperation master device. The *FMA* algorithm is continuously enabled, but only generates non-zero  $\dot{X}_F$  when the manipulator is in contact with the environment. The next subsection presents the general *NTRFC* design procedure, followed by modeling, *NTRFC* design, and simulation for the 1-dof 1*P* device.

## 3.1 NTRFC Design Procedure

The following *NTRFC* design procedure may be used for simulation and/or hardware implementation of a teleoperated manipulator system.

- 1) Design the manipulator controller to yield good, stable performance in free resolved-rate motion throughout the entire workspace. Any method may be used, including linearized independent PID joint control or feedback linearization with an optimal LQR controller.
- 2) Subject to the same manipulator free-motion controller structure and gains from above, determine FMA diagonal matrix gains  $K_F$  to ensure stable, desired transient and steady-state performance of the contact wrench given teleoperated Cartesian rate step inputs.
- 3) Determine system performance, sensitivity, and stability ranges. Simulate results. Implement in hardware. Iterate if necessary.

## 3.2 1P NTRFC Modeling

In this subsection we demonstrate the *NTRFC* concept and design procedure via a simple 1-dof device consisting of one prismatic joint (P). The 1P diagram is presented in Fig. 3. The device is modeled as a lumped mass m with viscous friction coefficient  $c_A$ . The relative device/environment compliance is modeled as a spring  $k_E$  and viscous damper  $c_E$ . The variable actuator length  $L_A$  operates along the X axis. The fixed length  $L_0$  gives the distance along X from the origin to the undisplaced environment location. The Cartesian environment displacement variable x is measured from the end of  $L_0$ .

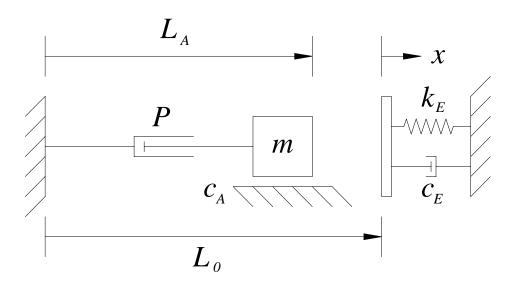


Figure 3. 1-dof 1P Device

The 1*P* device has trivial resolved-rate ( $J = J^{-1} = 1$ ) and forward kinematics ( $L_A$  is the total displacement along *X*) solutions. No rotations are possible. The dynamics equation of motion is:

$$f_A + f_M = m\ddot{L}_A + c_A\dot{L}_A \tag{1}$$

where  $f_A$  and  $f_M$  are the actuator and Cartesian MRF contact forces. The Coriolis and centrifugal terms are zero for this 1-dof example.

The 1*P NTRFC* block diagram is given in Fig. 4 (in the Laplace domain).  $\dot{x}_C$  is the modeled input from the human operator. Actual actuator length  $L_A$  is achieved by controller  $G_C$  servoing on length error  $L_E = L_C - L_A$ . In free motion,  $L_A$  is the output,  $-L_0$  is ignored, and  $f_M$  is zero so the scalar *FMA* gain  $k_F$  has no effect. At the instant of contact we assume connection of the mass to the environment. In contact with the environment the input  $\dot{x}_C$  exerts a force  $f_M$  on the environment after the transient dies out. Length  $L_0$  is included after contact in simulation to provide the environment displacement variable  $x = L_A - L_0$ . In simulation, x = 0 at contact which means that  $f_M$  is still 0 and thus the *FMA* feedback command  $\dot{x}_F$  is still 0. However, after contact, x will be non-zero and hence  $f_M$  and  $\dot{x}_F$  will also be non-zero. In contact, both x and  $f_M$  are output variables of interest. The controller (a *PD* structure is chosen), 1*P* dynamics, and environment transfer functions are assumed to be linear:

$$G_C = k_P + k_D s$$
  $G = \frac{1}{ms^2 + c_A s}$   $G_E = -c_E s - k_E$  (2)

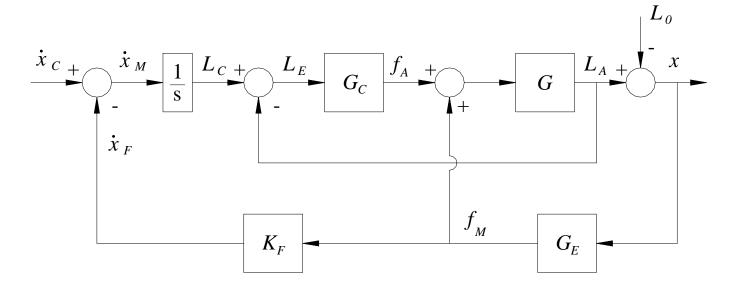


Figure 4. 1P NTRFC Block Diagram (no switches required)

The *NTRFC* design problem is stated: Given desired free-motion characteristics and desired contact force transient performance, calculate gains  $k_P$ ,  $k_D$ , and then  $k_F$ . First, the *PD* controller is designed for good free motion of  $L_A$  given  $L_C$  from the resolved rate commands. The unconstrained  $(f_M=0)$  transfer function relating output  $L_A$  to input  $L_C$  is:

$$T(s) = \frac{L_A(s)}{L_C(s)} = \frac{k_D s + k_P}{m s^2 + (c_A + k_D) s + k_P}$$
(3)

We specify 6% overshoot and 0.25 sec settling time for  $L_A$  in free motion; the desired characteristic polynomial is  $s^2 + 80s + 3359.6$ . It is difficult to calculate  $k_P$  and  $k_D$  via parameter matching due to the zero added by the PD controller. Therefore, we use an iterative approach to determine gains to satisfy the desired transient performance specifications; the result is  $k_P = 500$  and  $k_D = 75$ . This concludes step one of the NTRFC design procedure.

Step two requires determination of FMA gain  $k_F$  for good contact force transient characteristics (the transition between rate and force control), given the same  $k_P$  and  $k_D$  gains. The in-contact transfer function relating output  $f_M$  to input  $\dot{x}_C$  is now derived. Since Fig. 4 has two signals coming into the diagram ( $\dot{x}_C$  and  $L_0$ ), we must use the linear superposition principle (Eq. 4) to find the total contact force output  $f_M$  given the rate input  $\dot{x}_C$  (with  $L_0$ =0) and the environment parameter  $-L_0$  (with  $\dot{x}_C$  = 0):

$$f_M = f_{M1} + f_{M2} = T_{\dot{X}} \dot{x}_C - T_L L_0 \tag{4}$$

$$T_{\dot{X}} = \frac{f_{M1}}{\dot{x}_C} = \frac{GG_CG_E}{s(1 + GG_C - GG_E) + GG_CG_Ek_F}$$

$$T_L = \frac{f_{M2}}{L_0} = \frac{-s(1 + GG_C)G_E}{s(1 + GG_C - GG_E) + GG_CG_Ek_F}$$
(5)

Substituting the transfer functions from Eq. 2 into Eq. 5, the closed-loop transfer function  $T_{\dot{X}}$  for contact is (with  $L_0$ =0):

$$T_{\dot{X}} = \frac{-\left[k_D c_E s^2 + (k_D k_E + k_P c_E) s + k_P k_E\right]}{a_3 s^3 + a_2 s^2 + a_1 s + a_0}$$
(6)

where:

$$\begin{aligned} a_3 &= m \\ a_2 &= c_A + c_E + k_D - k_D c_E k_F \\ a_1 &= k_E + k_P - \left( k_D k_E + k_P c_E \right) k_F \\ a_0 &= -k_P k_E k_F \end{aligned}$$

The steady-state contact force and environment displacements are found using the final value theorem (separately for step inputs of magnitudes  $\dot{x}_C$  and  $L_0$ ):

$$f_{M1_{SS}} = \lim_{t \to \infty} f_{M1}(t) = \lim_{s \to 0} s f_{M1}(s) = \lim_{s \to 0} s T_{\dot{X}} \frac{\dot{x}_C}{s} = \frac{\dot{x}_C}{k_F}$$
 (7)

Note this force is independent of the device and environment parameters. Similarly,  $f_{M2_{SS}} = 0$ . The steady-state force exerted on the environment by the 1P device after control has transitioned from rate to force and the corresponding steady state displacement (under the rate input command  $\dot{x}_C$ ) are given below:

$$f_{M_{SS}} = f_{M1_{SS}} + f_{M2_{SS}} = \frac{\dot{x}_C}{k_F}$$
  $x_{SS} = \frac{-f_{M_{SS}}}{k_E} = \frac{-\dot{x}_C}{k_E k_F}$  (8)

The stability of this system is determined via the Routh-Hurwitz criterion. Since the leading coefficient in the characteristic polynomial  $a_3=m$  is positive, all coefficients  $a_i$  must also be positive. Also, the ensuing entries in the first column of the Routh array must be positive. This leads to the following three conditions for stability:

$$\frac{k_D + c_E + c_A}{k_D c_E} > k_F$$

$$ak_F^2 + bk_F + c > 0$$

$$- k_P k_F k_F > 0$$
(9)

where:

$$a = k_D c_E (k_P c_E + k_D k_E)$$

$$b = m k_P k_E - (k_D + c_E + c_A) (k_P c_E + k_D k_E) - k_D c_E (k_P + k_E)$$

$$c = (k_D + c_E + c_A) (k_P + k_E)$$

From the third condition in Eq. 9, the system is stable for  $k_F < 0$  since  $k_P$  and  $k_E$  are always positive. The first condition is always met for  $k_F < 0$  since the left-hand-side is always positive. For all cases simulated (including the example below), the quadratic condition did not dominate (i.e.  $k_F < d$  or  $k_F > e$  where both roots d and e are positive). However, this quadratic condition must be checked for all new cases.

Having determined the stability conditions and steady-state output variables, we have guidance for choosing  $k_F$ :  $k_F$  must be negative for stability and can be chosen for desired force amplification according Eq. 8 (left). However, the value of  $k_F$  has a significant effect on the transient force response. Therefore, it is preferable to adjust the force amplification through scaling rate commands  $\dot{x}_C$  from the master device and to choose  $k_F$  for desired transient performance. This is demonstrated in the example below.

## 1P Simulation Example

Given the following parameters, determine  $k_F$  and simulate 1P NTRFC motion. Above, the PD controller gains were determined for free motion:  $k_P$ =500 and  $k_D$ =75. The assumed device and environment parameters are (standard SI units are used): m=1;  $c_A$ =0.4;  $k_E$ =1,000;  $c_E$ =1; and  $L_0$ =0.10. A Matlab Simulink model was developed based on Fig. 4 to simulate 1P NTRFC motion. The  $-L_0$  input is enabled only after contact. We varied  $k_F$  and studied the resulting transient performance. Three measures of transient performance were used:  $\pm$ 2% settling time ( $T_S$ ), percent overshoot (%O), and percent "wiggle" (%W). The first two are familiar, but must be obtained from the simulation data; second-order system formulas will not work since our system is third order, with two zeros. The third, percent wiggle, is a third order effect. It is defined to be the maximum one-cycle peak-to-peak amplitude, divided by the final value. This measure was instituted because some simulation results had zero percent overshoot, but significant "wiggle". For this example, Figs. 5 show these transient performance results. In each, the solid line represents the steady-state force results  $f_{M_{SS}}$ ; the steady-state displacement transient results  $x_{SS}$  are very similar.

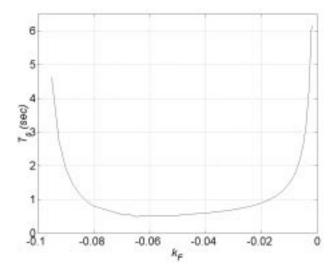


Figure 5a.  $T_S$  vs.  $k_F$ 

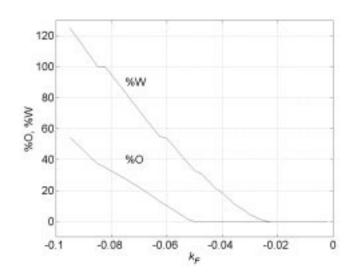
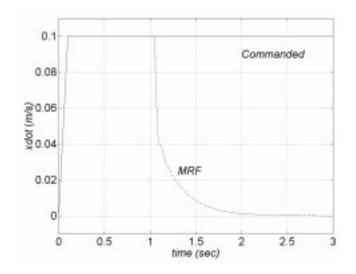


Figure 5b. %O and %W vs.  $k_F$ 

To determine the best  $k_F$  for in-contact transient force performance, polynomial approximations were made for the simulation data in Figs. 5. Then we used an optimization procedure to determine the  $k_F$  which minimizes the objective function:

$$f(k_F) = k_1 T S(k_F) + k_2 P O(k_F) + k_3 P W(k_F)$$
(10)

where  $k_i$  are the weighting factors giving the relative importance of settling time TS, percent overshoot PO, and percent wiggle PW. Percent wiggle is weighted more than overshoot; also given the difference in magnitude in settling time (relative to the two percent measures), settling time is weighted relatively heavier:  $k_1=75$ ,  $k_2=1$ , and  $k_3=3$ . The optimal value of  $k_F$  was determined in this manner to be  $k_F=-1$ 0.0225. The associated settling time is 0.82 sec, and percent overshoot and wiggle are both 0%. Using this value in the simulation, Figs. 6 show the NTRFC simulation results for this example. In this simulation, the input  $\dot{x}_C$  was linearly ramped from zero to a final value of 0.1 m/s (see Fig. 6a), to simulate a human operator commanding a constant rate starting from the master zero position. This figure also shows that the total resolved rate command  $\dot{x}_M$  goes to zero in the steady-state, after the FMA rate  $\dot{x}_F$  has been subtracted. As shown in Fig. 6b, the actuator length  $L_A$  increases linearly under rate control in free motion, experiences transient behavior (difficult to see at this scale), and assumes a steady-state value after the natural transition from rate to force control. Figure 6c shows that the contact forces  $f_M$  are zero in free motion until contact;  $f_M$  also experiences transient behavior and assumes a constant steady-state force after the transition even though the rate commands are still applied. The steady state values calculated from Eq. 8 agree with the simulation results:  $f_{M_{SS}} = 0.1/-0.0225 = -4.44$ and  $x_{SS} = 0.00444$ .



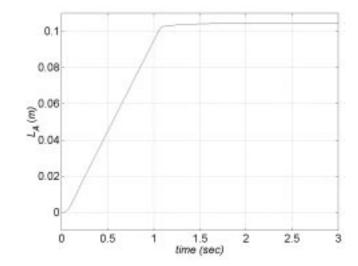


Figure 6a. NTRFC Rates

Figure 6b. Actuator Length  $L_A$ 

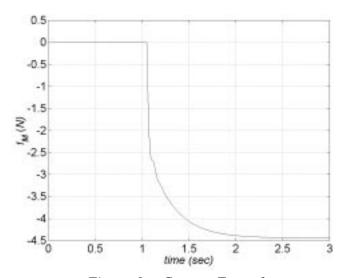


Figure 6c. Contact Force  $f_M$ 

This example demonstrates the *NTRFC* concept and simulated response. One weakness is that the environment parameters have a strong effect on the transient performance, though the stability condition  $k_F < 0$  is unchanged. We increased the environment parameters by an order of magnitude ( $k_E$ =10,000; and  $c_E$ =10) and obtained the new transient performance plots. These are similar in aspect to Figs. 5, but the  $k_F$  range is smaller, by a factor of 7.3 (note this is slightly less than the stiffness scaling of 10). With the stiffer environment, the percent overshoot and wiggle scales are unchanged. The settling

time is slighter larger for the stiffer environment. Using the same weighting factors in Eq. 10, we determined the optimal gain to be  $k_F = -0.0041$ . The associated settling time is 2.30 sec, and percent overshoot and wiggle are 0% and 24.1%, respectively. The *NTRFC* behavior is similar to that of Figs. 6, but the settling time is higher, there is more percent wiggle (it is zero in Fig. 6c), and the steady state exerted force is larger due to the lower  $k_F$ .

In *NTRFC* design we have the following tradeoff. From Eq. 8 (left), a higher  $k_F$  leads to a lower exerted force. However, a lower  $k_F$  leads to better transient performance, within bounds. Therefore, we need to balance these competing factors; a scaling factor on the operator rate input  $\dot{x}_C$  will help this.

In this simple example, the modeling is complex, and the transient performance is sensitive to  $k_F$  and the environment parameters. However, the exerted steady-state force is independent of the environment parameters. The NTRFC is implemented in a real-world system with a force-reflecting master in the next section.

## 4. PHANToM / Merlin Teleoperation with NTRFC

The *NTRFC* was implemented during summer 1998 in the Human Sensory Feedback (HSF) Laboratory of Wright-Patterson AFB. The master is a three-dof PHANToM haptic interface (Fig. 7) and the slave is a six-dof Merlin manipulator (Fig. 8). During summer 1997, the *NTRFC* was also implemented for the Merlin with a seven-dof force-reflecting exoskeleton master (Williams, 1997). The data below is from the PHANToM / Merlin system.



Figure 7. PHANToM

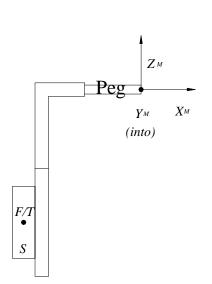


Figure 9. Merlin-Mounted Peg

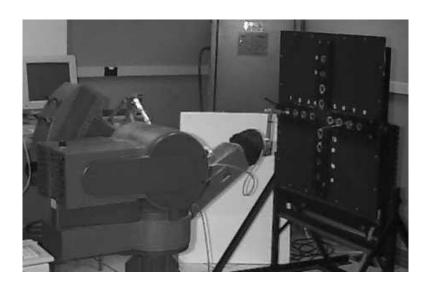


Figure 8. Merlin with Taskboard

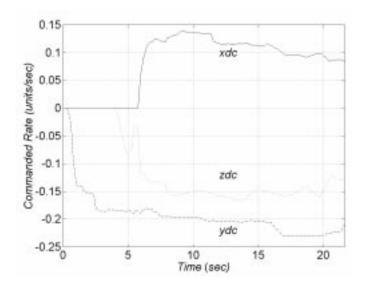


Figure 10. Split Operator View

The PHANToM used has three encoders and three motors, so only translational rates were commanded and only force (no moment) feedback was provided to the operator. The task involved pegin-the-hole insertions in a standard planar Fitts' law taskboard (Fitts and Peterson, 1964), with no need for orientation inputs. However, the full six-dof FMA algorithm was implemented, which provided automatic angular motion when moments were encountered due to misalignments. The Merlin-mounted peg is shown (with the F/T sensor which rigidly connects to the last manipulator link) in Fig. 9 and the Fitts' law taskboard is shown in Figs. 8 and 10. The L-shaped peg holder was designed for clear remote operator view. During the task,  $X_M$  is normal to the board, while  $Y_M$ - $Z_M$  is in the plane of the board. In the data presented, the Merlin was teleoperated via rate commands  $\dot{X}_C = \{v_C \ 0\}^T$  (the scaled operatordriven PHANToM position is taken to be  $v_C$ ). Data runs were performed with various contact scenarios. The data below is from a single typical run, complete with noise and non-steady signals of the real world. Starting from an initial configuration, the Merlin peg (Fig. 9) was flown in free rate motion until contact was established in each of the X, Y, and Z directions. The operator attempted to maintain constant  $\dot{X}_C$  during free and constrained phases of motion. The peg is mounted with a physical spring in the X axis, originally meant to protect against high forces in the insertion direction. Though this physical spring is no longer required with the software spring of the FMA algorithm, it was retained to demonstrate the NTRFC performs well with different environmental conditions (spring X and mostly rigid Y and Z).

Similar to the simulation example presented, it was found that the experimental system is stable for  $k_{Fx}$ ,  $k_{Fy}$ ,  $k_{Fz}$  < 0. However, unacceptable, though stable, transient contact transition can occur. The *NTRFC* gains are a 3x3 diagonal matrix  $K_F$ , tuned with equal diagonal components -0.0083. Figures 11 a, b, and c show the operator-commanded rates  $\dot{X}_C$ , the Cartesian displacements x, y, and z, and the Cartesian contact forces. The results are in English units. The Cartesian directions are shown in Fig. 9.

The operator attempts to hold constant rate inputs, starting at different times,  $v_C = \{0.1 - 0.2 - 0.15\}^T$ . The Cartesian displacements each display a constant velocity phase, followed by a transient transition, followed by a constant position phase. y is the sharpest, while x reaches its steady value gradually due to the hardware spring. The contact forces plot reveals all three directions contacted at nearly the same time (the peg was driven into a corner of the board, rather than a peg-hole). In free motion the forces are zero (or, noise and inertial loadings are within the deadband). At contact each rises to meet an approximately constant value. To the operator, the motion and contact looked and felt very smooth, with no notion of the variation seen in Fig. 11c. These results demonstrate the natural transition from rate to force control: in free motion, the displacement of the user's finger is proportional to the manipulator Cartesian rate; in contact, the force of the user's finger on the PHANToM (held at the same constant displacement) is proportional to the force exerted on the task board by the manipulator. No artificial controller switching is required. The steady-state force values  $f_{M_{SSY}} = 0.1/-0.0083 = -12;$ form (left): can be calculated with the of Eq.  $f_{M_{SSY}} = -0.2/-0.0083 = 24$ ; and  $f_{M_{SSY}} = -0.15/-0.0083 = 18$ . These values are achieved by experiment, as shown in Fig. 11c.



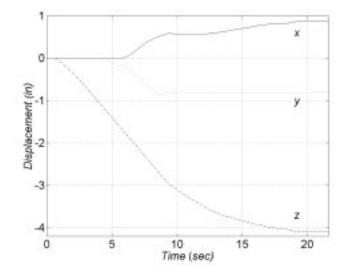


Figure 11a. Commanded Rates

Figure 11b. Cartesian Peg Displacements

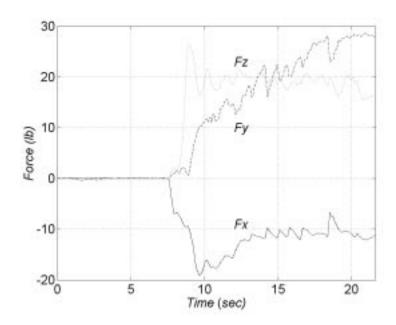
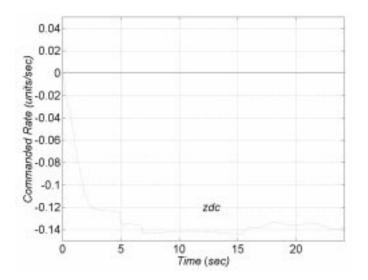


Figure 11c. Contact Forces

A second experimental run is shown in Figs. 12 a-c. This motion attempts to involve z only, although Figs. 12a and 12c show small x and y motions and forces also occur. The steady-state force value is  $f_{M_{SSY}} = -0.14/-0.0083 = 16.9$ , which agrees with the result.



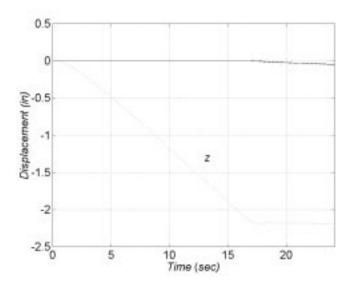


Figure 12a. z Commanded Rate

Figure 12b. z Cartesian Peg Displacement

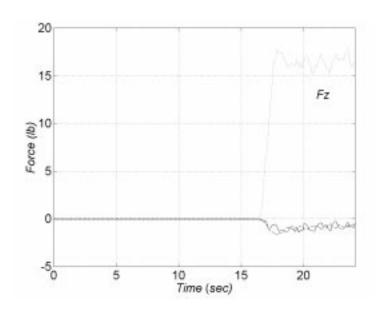


Figure 12c. z Contact Force

### 5. DISCUSSION

The objective of this article was to present a new, simple, promising control architecture for contacting the environment using rate control without controller mode changes. This section presents the limitations, strengths, and future work plans regarding the Naturally-Transitioning Rate-to-Force Controller (*NTRFC*) to help place this method in context with other alternatives.

## Limitations of the *NTRFC* and this Article

- Time-delayed teleoperation is not considered. If there is a significant time delay, the *FMA* algorithm can be implemented to run automatically at the remote site, which could help task completion. Force-reflection is not useful in the face of significant time delays.
- The *NTRFC* requires a good, continuous 6-axis Cartesian F/T sensor signal.
- This study considers relatively slow end-effector motions and relatively small loads outboard of the F/T sensor. In the lab, we found that there was no need to perform dynamics compensation on the F/T sensor reading due to end-effector/load inertia. Instead, a small F/T signal deadband handled this problem. However, the *NTRFC* architecture certainly admits this compensation if required.
- In the laboratory, the FMA feedback gain matrix  $K_F$  is tuned by trial and error. Our theoretical modeling gave us a good start in this process, but the effect of different manipulators and environments on the choice of  $K_F$  is not known yet.
- No human operator performance data is presented in this article. Instead we focus on presenting system effectiveness data for the *NTRFC* with a specific manipulator and environment in the lab.
- There is no specific comparison of the *NTRFC* with alternative contact transition methods in this article. A subjective comparison: according to our literature review, this is the ONLY method which allows rate control in free motion and force control in contact without the need for controller mode switches or the need to know the exact moment of contact. We claim these are very nice features without further comparisons to other methods.

#### Strengths of the *NTRFC* (in practice, not simulation)

- Simple to implement.
- No manipulator dynamics model required.
- No environment model required.
- No controller switching required.
- Don't need to determine the exact moment of contact.
- Good subjective "feel" in the lab intuitive and effective method according to various operators.
- Allows rate/contract transition "naturally".

#### Future Work Regarding the NTRFC

- Human/system teleoperation effectiveness study; compare with other control algorithms.
- Evaluate the *NTRFC* for different manipulators and environments.
- Develop and implement adaptive on-line NTRFC gain tuning.

## 6. CONCLUSION

This article has presented the Naturally-Transitioning Rate-to-Force Controller (*NTRFC*) for teleoperation of a manipulator. Existing rate control methods are often preferable to inverse position control in free motion, but unacceptable in contact with the environment due to integration of commands while under motion constraints, which builds up high contact wrenches. The *NTRFC* provides a method by which rate control in free motion naturally transitions to force control in contact. A force/moment accommodation (*FMA*) algorithm is the key ingredient. A wrist-mounted F/T sensor is required on the manipulator. The transition occurs due to the combination of the rate and *FMA* algorithms acting simultaneously on all Cartesian axes. No artificial control mode or gain changes are necessary, hence the method is insensitive to knowing the exact moment of contact in hardware implementation.

When a manipulator is teleoperated using the *NTRFC*, the following behavior results. In free motion, the displacement of the operator's hand with the master is proportional to the manipulator Cartesian rate. In contact, the displacement of the operator's hand is proportional to the Cartesian wrench exerted by the manipulator on the environment (control has transitioned naturally from rate to force). Furthermore, if the hand controller enables force-reflection to the operator, the wrench of the operator's hand reacting to the master force-reflection is proportional to the Cartesian wrench exerted by the manipulator on the environment.

A 1-dof 1*P* simulation example was presented to demonstrate the *NTRFC* concept. Then experimental results were presented from *NTRFC* implementation in the Human Sensory Feedback Laboratory of Wright-Patterson AFB. The experimental data (telerobotic system performance, not human operator performance) demonstrated that the *NTRFC* is working properly in practice.

## **ACKNOWLEDGMENTS**

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